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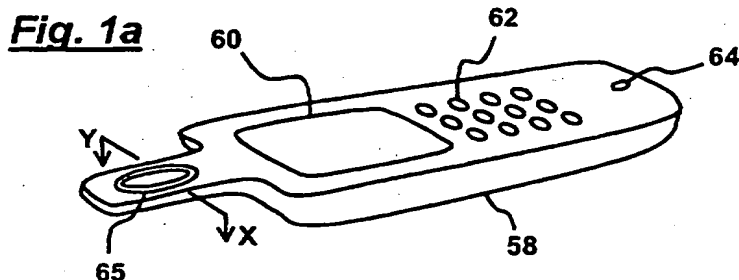
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(54) Abstract Title
Communication device using bone conduction

(57) A communication device 58 comprising a microphone 64, a conduction interface 65 and an electromechanical force transducer (112, fig 1b) mounted to the conduction interface 65 to drive the interface to conduct sound to a user by bone conduction. The transducer (112, fig 1b) has an intended operative frequency range and comprises a resonant element having a frequency distribution of modes in the operative frequency range and coupling means for mounting the transducer to the interface. The vibrations are transmitted through the mastoid bone of the user. The device may be used in a mobile phone, a hearing aid (fig 2) or a headset (fig 3). The hearing aid may be installed in a pair of spectacles (fig 2a).



At least one drawing originally filed was informal and the print reproduced here is taken from a later filed formal copy.

The claims were filed later than the filing date but within the period prescribed by Rule 25(1) of the Patents Rules 1995.

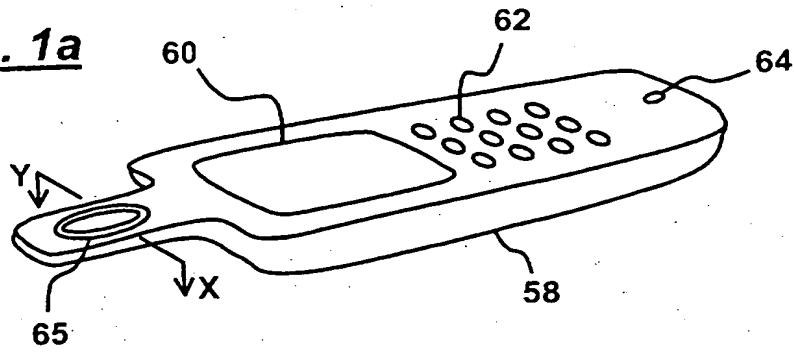
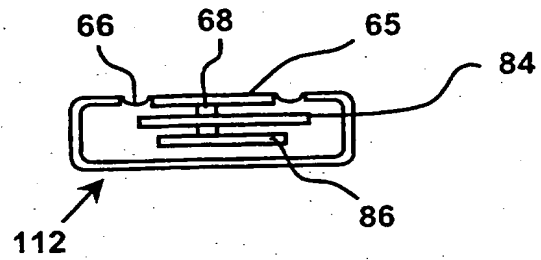
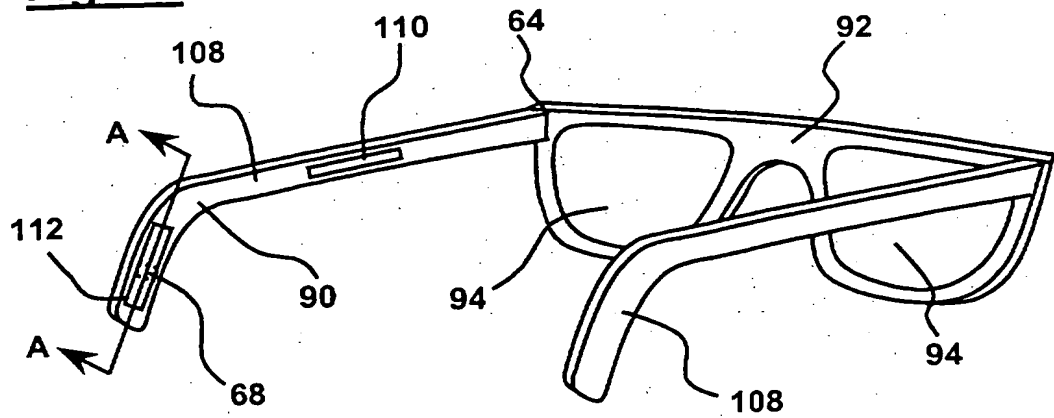
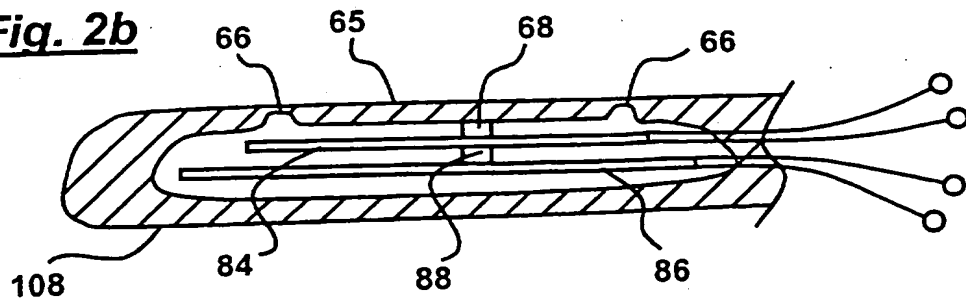
Fig. 1a**Fig. 1b****Fig. 2a****Fig. 2b**

Fig. 3

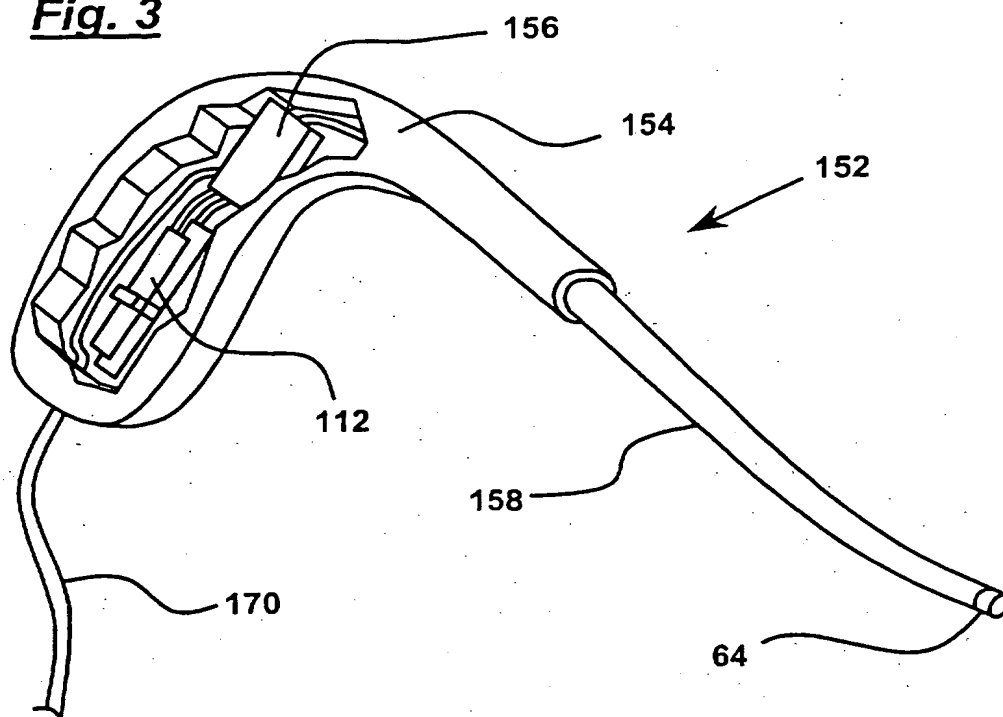


Fig. 4

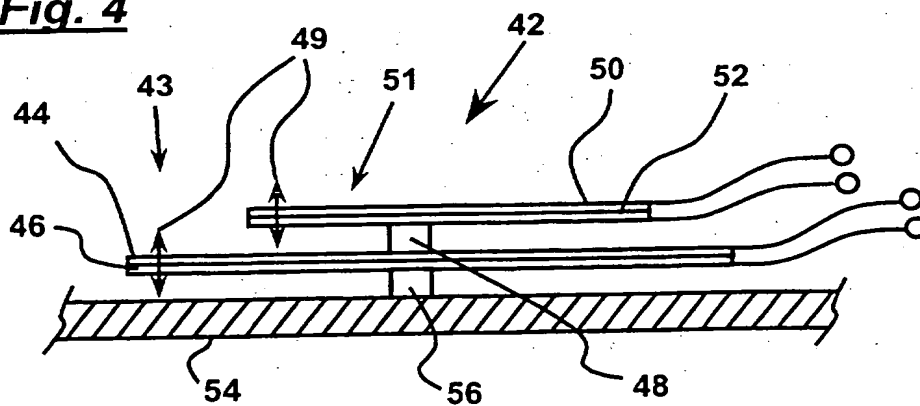


Fig. 5

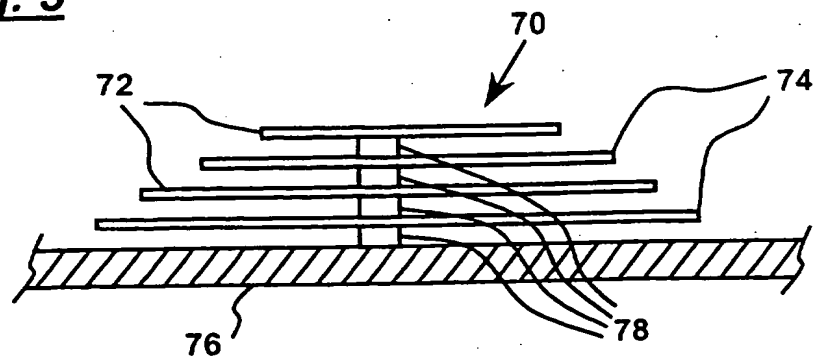


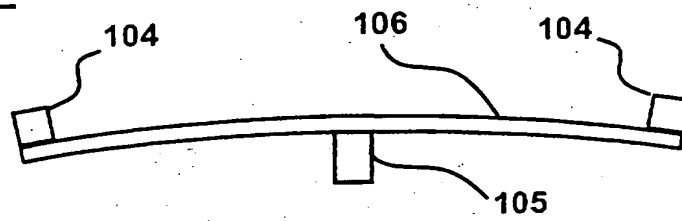
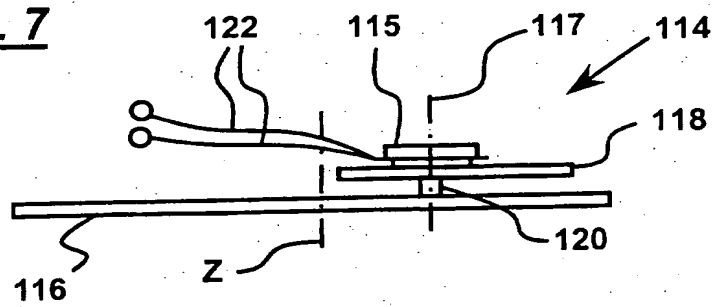
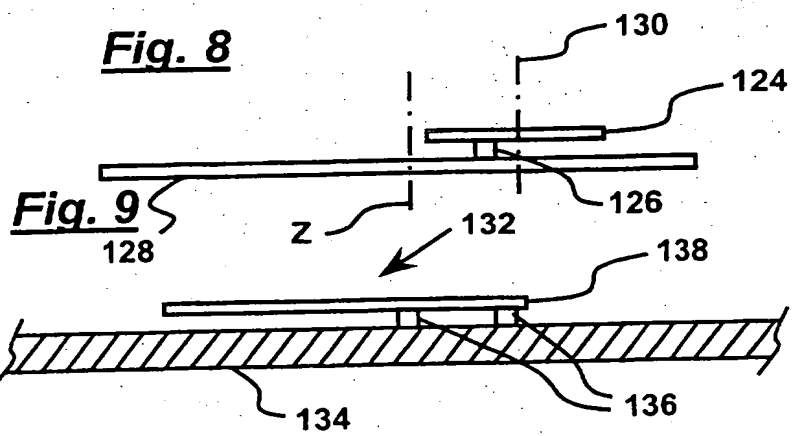
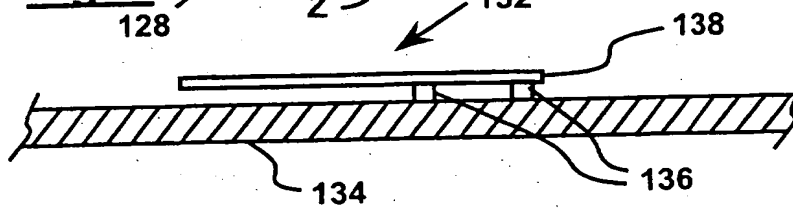
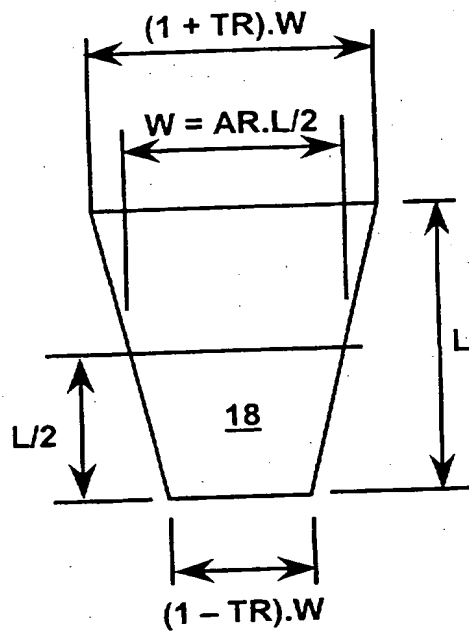
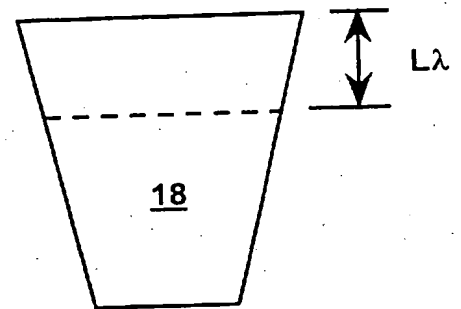
Fig. 6**Fig. 7****Fig. 8****Fig. 9**

Fig. 13a**Fig. 13b**

5 TITLE: COMMUNICATION DEVICE USING BONE CONDUCTION

The invention relates to communication devices such as
10 telephones, headsets or hearing aids which use bone conduction to transmit sound to a user.

Bone conduction is an established technique which enables deaf people to communicate. Typical examples comprise a transducer which is mounted adjacent a user's ear
15 so as to transmit sound to a user via conduction through the mastoid bone. However, known devices suffer from drawbacks such as excess pressure on the skull and the necessity of a heavy transducer means that their use is limited. For example, known hearing devices are currently manufactured by HiSonic and Viennatone but both devices use large
20 and heavy transducers. Thus the devices are somewhat uncomfortable and cumbersome.

Traditional headsets have an earpiece and microphone connected together and thus care must be taken to avoid
25 acoustical feedback. Hearing devices also suffer from this common problem of local feedback.

Similarly, in all telecommunications applications, the handset must comprise a microphone and an earpiece since it is necessary to transmit speech from a user to a microphone and onwards to a remote listener. A similar process occurs
5 for communication to be transmitted from the remote listener to the user. Thus, if there is any acoustical coupling between the microphone and earpiece in the handset, some form of feedback through the transmission network will occur. Accordingly, a normally fully duplex telecomms network
10 has to switch between different modes to combat this problem.

It is an object of the invention to provide improved communications device in which these known problems of weight and feedback are alleviated.

15 According to the invention, there is provided a communication device comprising a microphone, a conduction interface and an electromechanical force transducer mounted to the conduction interface to drive the interface to conduct sound to a user by bone conduction, characterised in
20 that the transducer has an intended operative frequency range and comprises a resonant element having a frequency distribution of modes in the operative frequency range and coupling means on the resonant element for mounting the transducer to the interface. In use, the conduction interface
25 may be mounted adjacent a user's mastoid bone.

The resonant element may be active e.g. may be a piezoelectric transducer and may be in the form of a strip of piezoelectric material. Alternatively, the resonant element may be passive and the transducer may further comprise an active transducer, e.g. an inertial or grounded vibration transducer, actuator or exciter, e.g. moving coil transducer. The active transducer may be a bender or torsional transducer (e.g. of the type taught in WO00/13464). Furthermore, the transducer may comprise combination of passive and active elements to form a hybrid transducer.

A number of transducer, exciter or actuator mechanisms have been developed to apply a force to a structure, e.g. an acoustic radiator of a loudspeaker. There are various types of these transducer mechanisms, for example moving coil, moving magnet, piezoelectric or magnetostrictive types. Typically, electrodynamic speakers using coil and magnet type transducers lose 99% of their input energy to heat whereas a piezoelectric transducer may lose as little as 1%. Thus, piezoelectric transducers are popular because of their high efficiency.

There are several problems with piezoelectric transducers, for example, they are inherently very stiff, for example comparable to brass foil, and are thus difficult to match to an acoustic radiator, especially to the air. Raising the stiffness of the transducer moves the fundamental resonant mode to a higher frequency. Thus such

piezoelectric transducers may be considered to have two operating ranges. The first operating range is below the fundamental resonance of the transducer. This is the "stiffness controlled" range where velocity rises with frequency and the output response usually needs equalisation. This leads to a loss in available efficiency. The second range is the resonance range beyond the stiffness range, which is generally avoided because the resonances are rather fierce.

Moreover, general teaching is to suppress resonances in a transducer, and thus piezoelectric transducers are generally used only used in the frequency range below or at the fundamental resonance of the transducers. Where piezoelectric transducers are used above the fundamental resonance frequency it is necessary to apply damping to suppress resonance peaks.

The problems associated with piezoelectric transducers similarly apply to transducers comprising other "smart" materials, i.e. magnetostrictive, electrostrictive, and electret type materials. Various piezoelectric transducers are also known, for example as described in EP 0993 231A of Shinsei Corporation, EP 0881 856A of Shinsei Corporation, US 4,593,160 OF Murata Manufacturing Co. Limited, US 4,401,857 of Sanyo Electric Co Limited, US 4,481,663 of Altec Corporation and UK patent application GB2,166,022A of

Sawafuji. However, it is an object of the invention to employ an improved transducer.

The transducer used in the present invention may be considered to be an intendedly modal transducer. The coupling means may be attached to the resonant element at a position which is beneficial for coupling modal activity of the resonant element to the interface. The parameters, e.g. aspect ratio, isotropy of bending stiffness, isotropy of thickness and geometry, of the resonant element may be selected to enhance the distribution of modes in the resonant element in the operative frequency range. Analysis, e.g. computer simulation using FEA or modelling, may be used to select the parameters.

The distribution may be enhanced by ensuring a first mode of the active element is near to the lowest operating frequency of interest. The distribution may also be enhanced by ensuring a satisfactory, e.g. high, density of modes in the operative frequency range. The density of modes is preferably sufficient for the active element to provide an effective mean average force which is substantially constant with frequency. Good energy transfer may provide beneficial smoothing of modal resonances. Alternatively, or additionally, the distribution of modes may be enhanced by distributing the resonant bending wave modes substantially evenly in frequency, i.e. to smooth peaks in the frequency response caused by "bunching" or clustering

of the modes. Such a transducer may thus be known as a distributed mode transducer or DMT.

Such an intendedly modal or distributed mode transducer, actuator, exciter or driver is described in International patent application PCT/GB01/00212 and US 09/768,002
5 filed January 24, 2001 (the latter of which is herein incorporated by reference).

The communication device may be a telephone having a casing. The casing may be formed with an extension remote
10 from the microphone, the transducer being mounted in the extension whereby in use the transducer is adjacent a user's mastoid bone. Thus, the user receives information from the structure of the device and thus communication is no longer airborne acoustic and the effects of acoustical
15 feedback should be reduced. Since no direct acoustical input to the ear canal is needed, a user may use an earplug with high attenuation, for example, for noisy environments.

Alternatively, the communication device may be a hearing aid, which may be mounted in a pair of spectacles or
20 may be disposable. Such hearing aids may provide sensory augmentation for blind people, for example by sensing the surroundings and providing signals which represent different surfaces and distance to the surfaces. Since the device should be free of feedback the location signals should not
25 be confused.

Alternatively, the communication device may be a headset, for example, for use in telephone call centres.

The advantages are that the transducer may be manufactured at very low cost and thus the communication devices
5 may be disposable. Furthermore, the transducers may be low weight and small, e.g. less than 3 grams but still have a high conversion efficiency. Thus they may be used in devices, e.g. in mobile telephones, where previous bone conduction arrangements are unsuitable, because of their
10 weight and bulkiness. The communication devices using intendedly modal transducers for bone conduction may have also reduced acoustic feedback.

The transducer may comprise a plurality of resonant elements each having a distribution of modes, the modes of
15 the resonant elements being arranged to interleave in the operative frequency range and thus enhance the distribution of modes in the transducer as a whole device. The resonant elements may have different fundamental frequencies and thus, the parameters, e.g. loading, geometry or bending
20 stiffness of the resonant elements may be different.

The resonant elements may be coupled together by connecting means in any convenient way, e.g. on generally stiff stubs, between the elements. The resonant elements are preferably coupled at coupling points which enhance the
25 modality of the transducer and/or enhance the coupling at the site to which the force is to be applied. Parameters

of the connecting means may be selected to enhance the modal distribution in the resonant element. The resonant elements may be arranged in a stack. The coupling points may be axially aligned.

5 The resonant element may be plate-like or may be curved out of planar. A plate-like resonant element may be formed with slots or discontinuities to form a multi-resonant system. The resonant element may be in the shape of a beam, trapezoidal, hyperelliptical or may be generally
10 disc shaped. Alternatively, the resonant element may be rectangular and may be curved out of the plane of the rectangle about an axis along the short axis of symmetry.

 The resonant element may be modal along two substantially normal axes, each axis having an associated
15 fundamental frequency. The ratio of the two fundamental frequencies may be adjusted for best modal distribution, e.g. 9:7 ($\sim 1.286:1$).

 As examples, the arrangement of such modal transducer may be any of: a flat piezoelectric disc; a combination of
20 at least two or preferably at least three flat piezoelectric discs; two coincident piezoelectric beams; a combination of multiple coincident piezoelectric beams; a curved piezoelectric plate; a combination of multiple curved piezoelectric plates or two coincident curved piezoelectric
25 beams.

The interleaving of the distribution of the modes in each resonant element may be enhanced by optimising the frequency ratio of the resonant elements, namely the ratio of the frequencies of each fundamental resonance of each resonant element. Thus, the parameter of each resonant element relative to one another may be altered to enhance the overall modal distribution of the transducer.

When using two active resonant elements in the form of beams, the two beams may have a frequency ratio (i.e. ratio of fundamental frequency) of 1.27:1. For a transducer comprising three beams, the frequency ratio may be 1.315:1.147:1. For a transducer comprising two discs, the frequency ratio may be 1.1 +/- 0.02 to 1 to optimise high order modal density or may be 3.2 to 1 to optimise low order modal density. For a transducer comprising three discs, the frequency ratio may be 3.03:1.63:1 or may be 8.19:3.20:1.

The parameters of the coupling means may be selected to enhance the distribution of modes in the resonant element in the operative frequency range. The coupling means may be vestigial, e.g. a controlled layer of adhesive.

The coupling means may be positioned asymmetrically with respect to the conduction interface so that the transducer is coupled asymmetrically. The asymmetry may be achieved in several ways, for example by adjusting the position or orientation of the transducer with respect to

axes of symmetry in the conduction interface or the transducer.

The coupling means may form a line of attachment. Alternatively, the coupling means may form a point or small local area of attachment where the area of attachment is small in relation to the size of the resonant element. The coupling means may be in the form of a stub and have a small diameter, e.g. 3 to 4 mm. The coupling means may be low mass.

The coupling means may comprise more than one coupling point and may comprise a combination of points and/or lines of attachment. For example, two points or small local areas of attachment may be used, one positioned near centre and one positioned at the edge of the active element. This may be useful for plate-like transducers which are generally stiff and have high natural resonance frequencies.

Alternatively only a single coupling point may be provided. This may provide the benefit, in the case of a multi-resonant element array, that the output of all the resonant elements is summed through the single coupling means so that it is not necessary for the output to be summed by the load. The coupling means may be chosen to be located at an anti-node on the resonant element and may be chosen to deliver a constant average force with frequency. The coupling means may be positioned away from the centre of the resonant element.

The position and/or the orientation of the line of attachment may be chosen to optimise the modal density of the resonant element. The line of attachment is preferably not coincident with a line of symmetry of the resonant element. For example, for a rectangular resonant element, the line of attachment may be offset from the short axis of symmetry (or centre line) of the resonant element. The line of attachment may have an orientation which is not parallel to a symmetry axis of the conduction interface.

10 The shape of the resonant element may be selected to provide an off-centre line of attachment which is generally at the centre of mass of the resonant element. One advantage of this embodiment is that the transducer is attached at its centre of mass and thus there is no inertial imbalance. This may be achieved by an asymmetric shaped resonant element which may be in the shape of a trapezium or trapezoid.

For a transducer comprising a beam-like or generally rectangular resonant element, the line of attachment may extend across the width of the resonant element. The area of the resonant element may be small relative to that of the acoustic radiator.

The operative frequency range may be over a relatively broad frequency range and may be in the audio range and/or ultrasonic range. Thus, operation over a range greater than the range defined by a single dominant, natural resonance

of the transducer may be achieved. The lowest frequency in the operative frequency range is preferably above a predetermined lower limit which is about the fundamental resonance of the transducer.

5

BRIEF DESCRIPTION OF DRAWINGS

The invention is diagrammatically illustrated, by way of example, in the accompanying drawings in which:

Figure 1A shows a schematic view of a mobile telephone embodying the present invention;

10 Figure 1B is a section along line A-A of Figure 1A;

Figure 2A shows a schematic view of a pair of spectacles embodying the present invention;

Figure 2B is a section along line A-A of Figure 2A;

15 Figure 3 shows a schematic view of a hearing aid/ head set embodying the present invention;

Figures 4 to 10 are side views of alternative modal transducers which may be used in the present invention;

Figure 11 is a plan view of an alternative modal transducer which may be used in the present invention;

20 Figure 12A is a schematic plan view of a parameterised model of a transducer which may be used in the present invention;

Figure 12B is a section perpendicular to the line of attachment of the transducer of Figure 12A;

Figure 13A is a schematic plan view of a parameterised model of a transducer which may be used in the present invention and

Figure 13B is a section perpendicular to the line of attachment of the transducer of Figure 13A.

Figures 1A and 1B show a mobile handset (58), e.g. for use as a mobile or cellular phone, which comprises a microphone (64), buttons (62) forming a keypad, and a screen (60). The microphone is mounted in one end of the handset and the opposed end of the handset (58) is formed with an extension. An intendedly modal transducer or distributed mode transducer as hereinbefore described and as described in PCT/GB01/00212 and US 09/768,002, is mounted in the extension so as to drive a conduction interface (65).

As shown more clearly in Figure 1B, the distributed mode transducer comprises upper and lower bimorph beams (84) and (86), the upper beam (84) being connected to the conduction interface (65) via a short stub (68) which extends across the width of the beams. The stub may be 1-2 mm wide and high and may be made from hard plastics and/or metal with suitable insulating layers to prevent electrical short circuits.

The beams are of unequal lengths with the upper beam (84) of length 36mm being longer than the lower beam (86) of length 32mm. Both beams have a width 7.5mm and a weight of 1.6 grams. Each beam consists of three layers, namely

two outer layers of piezoelectric ceramic material, e.g. PZT 5H, sandwiching a central brass vane. The outer layers may have a thickness of 150 microns and the central vane, a thickness of 100 microns. The outer layers may be attached
5 to the brass vane by adhesive layers which are typically 10 - 15 microns in thickness.

The conduction interface (65) is surrounded by a flexible surround (66), which supports the distributed mode transducer in the housing. The flexible surround prevents
10 too much structure borne vibration being introduced into the body of the mobile handset by the distributed mode transducer.

Figures 2A and 2B show a pair of spectacles (90) comprising a frame (92) supporting two lenses (94) and two
15 arms (108) which are hinged to the frame (92) and which support the spectacles on a user's ear (not shown). A microphone (64) is mounted in one arm (108) adjacent the frame (94) and an intended modal or distributed mode transducer (112) as hereinbefore described and as described
20 in PCT/GB01/00212 and US 09/768,002, is mounted in the same arm distanced from the microphone (64).

In use, the microphone (64) picks up the sound which is transmitted to and reproduced by the transducer without any airborne acoustic output. The transducer (112) is con-
25 nected to a coupling plate (65), which is suspended from the arm (108) by a suspension (66). The compliance of this

suspension is adjusted so that no structure borne vibration is transmitted through the arm (108) to the microphone (64). Thus a common problem of hearing aids, namely that of local feedback may be reduced.

5 As shown more clearly in Figure 2B, the distributed mode transducer (112) comprises upper and lower beams (84) and (86) joined by a stub (88). The upper beam (84) is connected to the conduction interface (65) via a short stub (68). The beams are of unequal lengths with the upper beam
10 (84) being longer than the lower beam (86). Wires are shown which connect the beams with an input source.

 An optional amplifier (110) is shown mounted in the same arm (108) at a position between the microphone (64) and the transducer (112). The amplifier may be powered by
15 a battery (not shown).

 Figure 3 shows an "over the ear" headset (152) which comprises a body (154) having an enclosure (156) which normally houses an amplifier, battery and signal processing and a cable (170) which feeds input and output signals to
20 the enclosure. A distributed mode transducer (112) as hereinbefore described and as described in PCT/GB01/00212 and US 09/768,002 is mounted in the body adjacent the enclosure (156). The transducer may be similar to those depicted in Figures 1B and 2B.

25 The body also comprises an extension (158) and a microphone (64) is mounted at an end of the extension which

is distal from the body. In use the enclosure would be behind a user's ear and the microphone is adjusted to be close to the speaker's mouth.

The remaining figures show alternative transducers which may be used in conjunction with the embodiments shown in Figures 1 to 3.

Figure 4 shows a transducer (42) which comprises a first piezoelectric beam (43) on the back of which is mounted a second piezoelectric beam (51) by connecting means in the form of a stub (48) located at the centre of both beams. Each beam is a bi-morph. The first beam (43) comprises two layers (44, 46) of piezoelectric material and the second beam (51) comprises two layers (50, 52). The poling directions of each layer of piezoelectric material are shown by arrows (49). Each layer (44, 50) has an opposite poling direction to the other layer (46, 52) in the bi-morph. The bimorph may also comprise a central conducting vane which allows a parallel electrical connection as well as adding a strengthening component to the ceramic piezoelectric layers. Each layer of each beam (44, 46) may be made of the same/different piezoelectric material. Each layer is generally of a different length.

The first piezoelectric beam (44, 46) is mounted on a structure (54), e.g. a conduction interface, by coupling means in the form of a stub (56) located at the centre of the first beam. By mounting the first beam at its centre

only the even order modes will produce output. By locating the second beam behind the first beam, and coupling both beams centrally by way of a stub they can both be considered to be driving the same axially aligned or co-incident position.

When elements are joined together, the resulting distribution of modes is not the sum of the separate sets of frequencies, because each element modifies the modes of the other. The two beams are designed so that their individual modal distributions are interleaved to enhance the overall modality of the transducer. The two beams add together to produce a useable output over a frequency range of interest. Local narrow dips occur because of the interaction between the piezoelectric beams at their individual even order modes.

The second beam may be chosen by using the ratio of the fundamental resonance of the two beams. If the materials and thicknesses are identical, then the ratio of frequencies is just the square of the ratio of lengths. If the higher f_0 (fundamental frequency) is simply placed half way between f_0 and f_1 of the other, larger beam, f_3 of the smaller beam and f_4 of the lower beam coincide.

Plotting a graph of a cost function against ratio of frequency for two beams shows that the ideal ratio is 1.27:1, namely where the cost function is minimised at point. This ratio is equivalent to the "golden" aspect ra-

tio (ratio of $f_{02}:f_{20}$) described in WO97/09482. The method of improving the modality of a transducer may be extended by using three piezoelectric beams in the transducer. The ideal ratio is 1.315:1.147:1.

5 The method of combining active elements, e.g. beams, may be extended to using piezoelectric discs. Using two discs, the ratio of sizes of the two discs depends upon how many modes are taken into consideration. For high order modal density, a ratio of fundamental frequencies of about
10 1.1 +/- 0.02 to 1 may give good results. For low order modal density (i.e. the first few or first five modes), a ratio of fundamental frequencies of about 3.2:1 is good. The first gap comes between the second and third modes of the larger disc.

15 Since there is a large gap between the first and second radial modes in each disc, much better interleaving is achieved with three rather than with two discs. When adding a third disc to the double disc transducer, the obvious first target is to plug the gap between the second and
20 third modes of the larger disc of the previous case. However, geometric progression shows that this is not the only solution. Using fundamental frequencies of f_0 , $\alpha.f_0$ and $\alpha^2.f_0$, and plotting $\text{rms}(\alpha, \alpha^2)$ there exist two principal optima for α . The values are about 1.72 and 2.90, with the

latter value corresponding to the obvious gap-filling method.

Using fundamental frequencies of f_0 , $\alpha.f_0$ and $\beta.f_0$ so that both scalings are free and using the above values of α as seed values, slightly better optima are achieved. The parameter pairs (α, β) are (1.63, 3.03) and (3.20, 8.19). These optima are quite shallow, meaning that variations of 10%, or even 20%, in the parameter values are acceptable.

An alternative approach for determining the different discs to be combined is to consider the cost as a function of the ratio of the radii of the three discs. The cost functions may be RSCD (ratio of sum of central differences), SRCD (sum of the ratio of central differences) and SCR (sum of central ratios) shown by lines (64), (66) and (68) respectively. For a set of modal frequencies, $f_0, f_1, f_2, \dots, f_N$, these functions are defined as:

RSCD (R sum CD) :

$$RSCD = \frac{1}{N-1} \sum_{n=1}^{N-1} \frac{(f_{n+1} + f_{n-1} - 2f_n)^2}{f_0}$$

20 SRCD (sum RCD) :

$$SRCD = \frac{1}{N-1} \sum_{n=1}^{N-1} \left(\frac{f_{n+1} + f_{n-1} - 2f_n}{f_n} \right)^2$$

CR:

$$SCR = \frac{1}{N-1} \sum_{n=1}^{N-1} \left(\frac{f_{n+1} \cdot f_{n-1}}{(f_n)^2} \right)$$

The optimum radii ratio, i.e. where the cost function is minimised, is 1.3 for all cost functions. Since the square of the radii ratio is equal to the frequency ratio, for these identical material and thickness discs, the results of $1.3 \times 1.3 = 1.69$ and the analytical result of 1.67 are in good agreement.

Alternatively or additionally, passive elements may be incorporated into the transducer to improve its overall modality. The active and passive elements may be arranged in a cascade. Figure 5 shows a multiple disc transducer (70) comprising two active piezoelectric elements (72) stacked with two passive resonant elements (74), e.g. thin metal plates so that the modes of the active and passive elements are interleaved. The elements are connected by connecting means in the form of stubs (78) located at the centre of each active and passive element. The elements are arranged concentrically. Each element has different dimensions with the smallest and largest discs located at the top and bottom of the stack, respectively. The transducer (70) is mounted on a load device (76), e.g. a conductive interface, by coupling means in the form of a stub (78) located at the centre of the first passive device which is the largest disc.

The method of improving the modality of a transducer may be extended to a transducer comprising two active elements in the form of piezoelectric plates. Two plates of dimensions (1 by α) and (α by α^2) are coupled at (3/7, 4/9).
5 The frequency ratio is therefore about 1.3:1 ($1.14 \times 1.14 = 1.2996$).

As shown in Figure 6, small masses (104) may be mounted at the end of the piezoelectric transducer (106) having coupling means (105). In Figure 7, the transducer
10 (114) is an inertial electrodynamic moving coil exciter, e.g. as described in WO97/09842, having a voice coil forming an active element (115) and a passive resonant element in the form of a modal plate (118). The active element (115) is mounted on the modal plate (118) and off-centre of
15 the modal plate. The modal plate (118) is mounted on the panel (116) by a coupler (120). The coupler is aligned with the axis (Z) of the active element but not with the axis (119) normal to the plane of the panel (116). Thus the transducer is not coincident with the normal axis (Z). The
20 active element is connected to an electrical signal input via electrical wires (122). The modal plate (118) is perforate to reduce the acoustic radiation therefrom and the active element is located off-centre of the modal plate (118), for example, at the optimum mounting position, i.e.
25 (3/7, 4/9).

Figure 8 shows a transducer (124) comprising an active piezoelectric resonant element which is mounted by coupling means (126) in the form of a stub to a conduction interface (128). Both the transducer (124) and conduction interface (128) have ratios of width to length of 1:1.13. The coupling means (126) is not aligned with any axes (130, X,Y,Z) of the transducer or the conduction interface. Furthermore, the placement of the coupling means is located at the optimum position with off-centre with respect to both the transducer (124) and the conduction interface (128).

Figure 9 shows a transducer (132) in the form of active piezoelectric resonant element in the form of a beam. The transducer (132) is coupled to a conduction interface (134) by two coupling means (136) in the form of stubs. One stub is located towards an end (138) of the beam and the other stub is located towards the centre of the beam.

Figure 10 shows a transducer (140) comprising two active resonant elements (142,143) coupled by connecting means (144) and an enclosure (148) which surrounds the connecting means (144) and the resonant elements (142). The transducer is thus made shock and impact resistant. The enclosure is made of a low mechanical impedance rubber or comparable polymer so as not to impede the transducer

operation. If the polymer is water resistant, the transducer (140) may be made waterproof.

The upper resonant element (142) is larger than the lower resonant element (143) which is coupled to a panel (145) via a coupling means in the form of a stub. The stub is located at the centre of the lower resonant element (143). The power couplings (150) for each active element extend from the enclosure to allow good audio attachment to a load device (not shown).

Figure 11 shows a transducer (160) in the form of an plate-like active resonant element. The resonant element is formed with slots (162) which define fingers (164) and thus form a multi-resonant system. The resonant element is mounted on a conduction interface (168) by a coupling means in the form of a stub (166).

In Figures 12A and 1B, the transducer (14) is rectangular with out-of-plane curvature and is a pre-stressed piezoelectric transducer of the type disclosed in US patent 5632841 (International patent application WO 96/31333) and produced by PAR Technologies Inc under the trade name NAS-DRIV. Thus the transducer (14) is an active resonant element. The transducer has width (W) and length (L) and the position (x) of the attachment point (16).

The curvature of the transducer (14) means that the coupling means (16) is in the form of a line of attachment. Thus the transducer (14) is attached to the panel (12) only

along line A-A. When the transducer is mounted along a line of attachment along the short axis through the centre, the resonance frequencies of the two arms of the transducer are coincident. The optimum suspension point may be modelled and is has the line of attachment at 43% to 44% along the length of the resonant element. The cost function (or measure of "badness") is minimised at this value; this corresponds to an estimate for the attachment point at $4/9$ ths of the length. Furthermore, computer modelling showed this attachment point to be valid for a range of transducer widths. A second suspension point at 33% to 34% along the length of the resonant element also appears suitable.

By plotting a graph of cost (or rms central ratio) against aspect ratio ($AR=W/2L$) for a resonant element mounted at 44% along its length, the optimum aspect ratio may be determined to be 1.06 ± 0.01 to 1 since the cost function is minimised at this value.

The optimum angle of attachment θ to the panel (12) may be determined using two "measures of badness" to find the optimum angle. For example, the standard deviation of the log (dB) magnitude of the response is a measure of "roughness". Such figures of merit/badness are discussed in International Application WO 99/41839, to the present applicants. For an optimised transducer, namely one with aspect ratio 1.06:1 and attachment point at 44% using modelling, rotation of the line of attachment (16) will have a

marked effect since the attachment position is not symmetrical. There is a preference for an angle of about 270° , i.e. with the longer end facing left.

Figures 13A and 13B show an asymmetrically shaped transducer (18) in the form of a resonant element having a trapezium shaped cross-section. The shape of a trapezium is controlled by two parameters, AR (aspect ratio) and TR (taper ratio). AR and TR determine a third parameter, λ , such that some constraint is satisfied - for example, equal mass either side of the line.

The constraint equation for equal mass (or equal area) is as follows;

$$\int_0^{\lambda} \left(1 + 2TR \left(\frac{1}{2} - \xi\right)\right) d\xi = \int_{\lambda}^1 \left(1 + 2TR \left(\frac{1}{2} - \xi\right)\right) d\xi$$

The above may readily be solved for either TR or λ as the dependent variable, to give:

$$TR = \frac{1 - 2\lambda}{2\lambda(1 - \lambda)} \quad \text{or} \quad \lambda = \frac{1 + TR - \sqrt{1 + TR^2}}{2TR} \approx \frac{1}{2} - \frac{TR}{4}$$

Equivalent expressions are readily obtained for equalising the moments of inertia, or for minimising the total moment of inertia.

The constraint equation for equal moment of inertia (or equal 2nd moment of area) is as follows;

$$\int_0^{\lambda} \left(1 + 2TR \left(\frac{1}{2} - \xi\right)\right) (\lambda - \xi)^2 d\xi = \int_{\lambda}^1 \left(1 + 2TR \left(\frac{1}{2} - \xi\right)\right) (\xi - \lambda)^2 d\xi$$

$$TR = \frac{(\lambda^2 - \lambda + 1)(2\lambda - 1)}{2\lambda^4 - 4\lambda^3 + 2\lambda - 1} \quad \text{or} \quad \lambda \approx \frac{1}{2} - \frac{TR}{8}$$

The constraint equation for minimum total moment of inertia
5 is

$$\frac{d}{d\lambda} \left(\int_0^1 \left(1 + 2TR \left(\frac{1}{2} - \xi \right) \right) (\lambda - \xi)^2 d\xi \right) = 0$$

$$TR = 3 - 6\lambda \quad \text{or} \quad \lambda = \frac{1}{2} - \frac{TR}{6}$$

10

A cost function (measure of "badness") was plotted for
the results of 40 FEA runs with AR ranging from 0.9 to
1.25, and TR ranging from 0.1 to 0.5, with λ constrained
for equal mass. The transducer is thus mounted at the cen-
15 tre of mass. The results are tabulated below and show that
there is an optimum shape with AR=1 and TR=0.3, giving λ at
close to 43%.

tr	λ	0.9	0.95	1	1.05	1.1	1.15	1.2	1.25
0.1	47.51%	2.24%	2.16%	2.16%	2.24%	2.31%	2.19%	2.22%	2.34%
0.2	45.05%	1.59%	1.61%	1.56%	1.57%	1.50%	1.53%	1.66%	1.85%
0.3	42.66%	1.47%	1.30%	1.18%	1.21%	1.23%	1.29%	1.43%	1.59%
0.4	40.37%	1.32%	1.23%	1.24%	1.29%	1.25%	1.29%	1.38%	1.50%
0.5	38.20%	1.48%	1.44%	1.48%	1.54%	1.56%	1.58%	1.60%	1.76%

One advantage of a trapezoidal transducer is thus that
20 the transducer may be mounted along a line of attachment
which is at its centre of gravity/mass but is not a line of
symmetry. Such a transducer would thus have the advantages
of improved modal distribution, without being inertially
unbalanced. The two methods of comparison used previously

again select 270° to 300° as the optimum angle of orientation.

The transducer used in the present invention may be seen as the reciprocal of a distributed mode panel, e.g. as described in WO97/09842, in that the transducer is designed to be a distributed mode object.

CLAIMS

1. A communication device comprising a microphone, a conduction interface and an electromechanical force transducer mounted to the conduction interface to drive the interface to conduct sound to a user by bone conduction, characterised in that the transducer has an intended operative frequency range and comprises a resonant element having a frequency distribution of modes in the operative frequency range and coupling means for mounting the transducer to the interface.

2. A communication device according to claim 1, wherein the conduction interface is mounted adjacent a user's mastoid bone.

3. A communication device according to claim 1 or claim 2, wherein the coupling means is attached to the resonant element at a position which is beneficial for coupling modal activity of the resonant element to the interface.

4. A communication device according to any one of the preceding claims, wherein the parameters, e.g. aspect ratio, isotropy of bending stiffness, isotropy of thickness and geometry, of the resonant element are selected to enhance the distribution of modes in the resonant element in the operative frequency range.

5. A communication device according to claim 4, wherein the distribution is enhanced by ensuring a first mode of the active element is near to the lowest operating frequency of interest.

5 6. A communication device according to any one of claims 1 to 3, wherein the resonant element is active.

7. A communication device according to any one of claims 1 to 5, wherein the resonant element is passive and the transducer further comprises an active element.

10 8. A communication device according to any one of claims 1 to 5, wherein the transducer comprises a combination of passive and active elements to form a hybrid transducer.

9. A communication device according to any one of
15 claims 6 to 8, wherein the distribution of modes in the resonant element is enhanced by ensuring the distribution has a density of modes which is sufficient for the active element to provide an effective mean average force which is substantially constant with frequency.

20 10. A communication device according to any one of claims 4, 5 or 9, wherein the distribution of modes is enhanced by distributing the resonant bending wave modes substantially evenly in frequency.

11. A communication device according to any preceding
25 claim, in the form of a telephone having a casing.

12. A communication device according to claim 11,
wherein the casing is formed with an extension remote from
the microphone, the transducer being mounted in the
extension whereby in use the transducer is adjacent a
5 user's mastoid bone.

13. A communication device according to any one of
claims 1 to 10, in the form of a hearing aid.

14. A communication device according to claim 13,
which is mounted in a pair of spectacles.

10 15. A communication device according to any one of
claims 1 to 10, in the form of a headset.

16. A communication device according to any one of
the preceding claims, wherein the transducer comprises a
plurality of resonant elements each having a distribution
15 of modes, the modes of the resonant elements being arranged
to interleave in the operative frequency range whereby the
distribution of modes in the transducer as a whole device
is enhanced.

17. A communication device according to claim 16,
20 wherein the resonant elements are coupled together by
connecting means between the elements.

18. A communication device according to claim 16 or
claim 17, wherein the resonant elements are coupled at
coupling points which enhance the modality of the
25 transducer.

19. A communication device according to claim 18, wherein the coupling points are axially aligned.

20. A communication device according to any one of the preceding claims, wherein the resonant element is
5 plate-like.

21. A communication device according to claim 20, wherein the plate-like resonant element is formed with slots to form a multi-resonant system.

22. A communication device according to any one of
10 the preceding claims, wherein the shape of the resonant element is selected from the group consisting of beam-like, trapezoidal, hyperelliptical, generally disc shaped or rectangular.

23. A communication device according to any one of
15 the preceding claims, wherein the resonant element is modal along two substantially normal axes, each axis having an associated fundamental frequency and the ratio of the two associated fundamental frequencies being adjusted for best modal distribution.

20 24. A communication device according to claim 23, wherein the ratio of the two fundamental frequencies is 9:7.

25 25. A loudspeaker according to any one of the preceding claims, wherein the transducer comprises more than one resonant element each having a fundamental resonance frequency, the distribution of the modes in each

resonant element being enhanced by optimising a frequency ratio of each fundamental resonance frequency.

26. A communication device according to claim 25, comprising two active resonant elements in the form of beams, the two beams having a frequency ratio of 1.27:1.

27. A communication device according to claim 25, comprising three resonant elements in the form of beams, the beams having a frequency ratio of 1.315:1.147:1.

28. A communication device according to claim 25, comprising two resonant elements in the form of discs, the discs having a frequency ratio of 1.1 ± 0.02 to 1, whereby high order modal density is optimised.

29. A communication device according to claim 25, comprising two resonant elements in the form of discs, the discs having a frequency ratio of 3.2 to 1, whereby low order modal density is optimised.

30. A communication device according to claim 25, comprising three resonant elements in the form of discs, the discs having a frequency ratio of 3.03:1.63:1.

31. A communication device according to any one of the preceding claims, wherein the coupling means is positioned asymmetrically with respect to the conduction interface so that the transducer is coupled asymmetrically.

32. A communication device according to any one of the preceding claims, wherein the coupling means forms a small local area of attachment, the area of attachment

being small in relation to the size of the resonant element.

33. A communication device according to any preceding claim, wherein the coupling means is chosen to be located at an anti-node on the resonant element.



INVESTOR IN PEOPLE

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Claims searched: All

Examiner: Paul Marshall
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Patents Act 1977 Search Report under Section 17

Databases searched:

UK Patent Office collections, including GB, EP, WO & US patent specifications, in:

UK Cl (Ed.T):

Int Cl (Ed.7): H04M 1/02 H04R 25/00, 25/02, 25/04

Other: Online: EPODOC, WPI, JAPIO

Documents considered to be relevant:

Category	Identity of document and relevant passage	Relevant to claims
A	WO 01/28195 A1 (BOESEN) See figures.	-
A	WO 01/97489 A1 (KIM) See figures.	-
A	US 5448637 A (ELECTRONICS & TELECOM RES INST) See figures.	-
A	US 4150262 A (ONO) See figures.	-

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Y	Document indicating lack of inventive step if combined with one or more other documents of same category.	P	Document published on or after the declared priority date but before the filing date of this invention.
&	Member of the same patent family	E	Patent document published on or after, but with priority date earlier than, the filing date of this application.